

Variable Stiffness Actuation: Modeling and Control

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Why Variable Stiffness Actuation?

Improves the safety of the robotic device with respect to:

- interaction with unknown environment
- unexpected collisions
- limited controller and sensors bandwidth
- actuator failures

A. Bicchi and G. Tonietti. “Fast and soft arm tactics: Dealing with the safety-performance trade-off in robot arms design and control”. IEEE Robotics and Automation Magazine, 2004.

G. Tonietti, R. Schiavi, and A. Bicchi. “Design and control of a variable stiffness actuator for safe and fast physical human/robot interaction”. In Proc. IEEE Int. Conf. on Robotics and Automation, 2005.



Why Variable Stiffness Actuation?

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Drawbacks of the Variable Stiffness Actuation:

- A more complex mechanical design
- The number of actuators increases
- Non-linear transmission elements must be used
- High non-linear and cross coupled dynamic model



Dynamic Model of Robots with Variable Joint Stiffness

- Robot dynamic equations

$$\begin{aligned}M(q) \ddot{q} + N(q, \dot{q}) + K(q - \theta) &= 0 \\ B \ddot{\theta} + K(\theta - q) &= \tau\end{aligned}$$

- The diagonal joint stiffness matrix is considered time-variant

$$K = \text{diag}\{k_1, \dots, k_n\}, \quad K = K(t) > 0$$

- Alternative notation

$$K(q - \theta) = \Phi k, \quad \Phi = \text{diag}\{(q_1 - \theta_1), \dots, (q_n - \theta_n)\}, \quad k = [k_1, \dots, k_n]^T$$

- ① The joint stiffness k can be directly changed by means of a (suitably scaled) additional command τ_k

$$k = \tau_k$$

- ② The variation of joint stiffness may be modeled as a second-order dynamic system

$$\ddot{k} = \phi(x, k, \dot{k}, \tau_k)$$



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Dynamic Model of Robots with Variable Joint Stiffness

- The input u and the robot state x are:

$$u = \begin{bmatrix} \tau \\ \tau_k \end{bmatrix} \in \mathbb{R}^{2n}, \quad x = [q^T \quad \dot{q}^T \quad \theta^T \quad \dot{\theta}^T]^T \in \mathbb{R}^{4n}$$

- In the case of second-order stiffness variation model, the state vector of the robot becomes:

$$x_e = [q^T \quad \dot{q}^T \quad \theta^T \quad \dot{\theta}^T \quad k^T \quad \dot{k}^T]^T \in \mathbb{R}^{6n}$$

- In all cases, the objective will be to simultaneously control the following set of outputs

$$y = \begin{bmatrix} q \\ k \end{bmatrix} \in \mathbb{R}^{2n}$$

namely the link positions (and thus, through the robot direct kinematics, the end-effector pose) and the joint stiffness



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Dynamic Inversion

- The motion is specified in terms of a desired smooth position trajectory $q = q_d(t)$ and joint stiffness matrix $K = K_d(t)$ (or, equivalently, of the vector $k = k_d(t)$)
- Assuming $k = \tau_k$, we have simply $\tau_{k,d} = k_d(t)$ and only the computation of the nominal motor torque τ_d is of actual interest
- The robot dynamic equation is differentiated twice with respect to time

$$M(q) \ddot{q} + \dot{M}(q) \dot{q} + \dot{N}(q, \dot{q}) + \dot{K}(q - \theta) + K(\dot{q} - \dot{\theta}) = 0$$

and

$$M(q) \ddot{\ddot{q}} + 2\dot{M}(q) \ddot{q} + \ddot{M}(q) \dot{q} + \ddot{N}(q, \dot{q}) + K(\ddot{q} - \ddot{\theta}) + 2\dot{K}(\dot{q} - \dot{\theta}) + \ddot{K}(q - \theta) = 0$$



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Dynamic Inversion

- Reference motor position along the desired robot trajectory

$$\theta_d = q_d + K_d^{-1} (M(q_d)\ddot{q}_d + N(q_d, \dot{q}_d)).$$

- Reference motor velocity

$$\begin{aligned} \dot{\theta}_d = \dot{q}_d + K_d^{-1} \left(M(q_d)q_d^{[3]} + \dot{M}(q_d)\ddot{q}_d + \dot{N}(q_d, \dot{q}_d) \right. \\ \left. - \dot{K}_d K_d^{-1} (M(q_d)\ddot{q}_d + N(q_d, \dot{q}_d)) \right). \end{aligned}$$

- Actuators dynamic model inversion

$$\ddot{\theta} = B^{-1} [\tau - K(\theta - q)],$$



Actuator Torques Computation

- Reference motor torque along the desired trajectory

$$\tau_d = M(q_d)\ddot{q}_d + N(q_d, \dot{q}_d) + BK_d^{-1}\alpha_d \left(q_d, \dot{q}_d, \ddot{q}_d, q_d^{[3]}, q_d^{[4]}, k_d, \dot{k}_d, \ddot{k}_d \right)$$

- Some minimal smoothness requirements are imposed

$$q_d(t) \in \mathbb{C}^4 \quad \text{and} \quad k_d(t) \in \mathbb{C}^2$$

- Discontinuous models of friction or actuator dead-zones on the motor side can be considered without problems
- Discontinuous phenomena acting on the link side should be approximated by a smooth model
- The command torques τ_d can be kept within the saturation limits by a suitable time scaling of the manipulator trajectory



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Second-Order Stiffness Model

- The dynamics of the joint stiffness k is written as a generic nonlinear function of the system configuration

$$\ddot{k} = \beta(\mathbf{q}, \theta) + \gamma(\mathbf{q}, \theta) \tau_k$$

- Double differentiation wrt time of the robot dynamics

$$\begin{aligned} M \mathbf{q}^{[4]} + 2 \dot{M} \mathbf{q}^{[3]} + \ddot{M} \ddot{\mathbf{q}} + \ddot{N} \\ + K (\ddot{\mathbf{q}} - B^{-1} [\tau - K(\theta - \mathbf{q})]) \\ + 2 \dot{K} (\dot{\mathbf{q}} - \dot{\theta}) + \Phi (\beta + \gamma \tau_k) = 0 \end{aligned}$$

where both the inputs τ and τ_k appear

- Important notes

$$\ddot{\mathbf{q}} = \ddot{\mathbf{q}}(\dot{\mathbf{q}}, \mathbf{q}), \quad \mathbf{q}^{[3]} = \mathbf{q}^{[3]}(\dot{\mathbf{q}}, \mathbf{q}), \quad \mathbf{q}^{[4]} = \mathbf{q}^{[4]}(\dot{\mathbf{q}}, \mathbf{q})$$

Feedback Linearized Model

- The overall system can be written as

$$\begin{bmatrix} q^{[4]} \\ \ddot{k} \end{bmatrix} = \begin{bmatrix} \alpha(x_e) \\ \beta(q, \theta) \end{bmatrix} + Q(x_e) \begin{bmatrix} \tau \\ \tau_k \end{bmatrix}$$

where $Q(x_e)$ is the decoupling matrix:

$$Q(x_e) = \begin{bmatrix} M^{-1}KB^{-1} & M^{-1}\Phi \gamma(q, \theta) \\ 0_{n \times n} & \gamma(q, \theta) \end{bmatrix}$$

- Non-Singularity Conditions

$$\left. \begin{array}{l} k_i > 0 \\ \gamma_i(q_i, \theta_i) \neq 0 \end{array} \right\} \forall i = 1, \dots, n$$

- By applying the static state feedback

$$\begin{bmatrix} \tau \\ \tau_k \end{bmatrix} = Q^{-1}(x_e) \left(- \begin{bmatrix} \alpha(x_e) \\ \beta(q, \theta) \end{bmatrix} + \begin{bmatrix} v_q \\ v_k \end{bmatrix} \right)$$

the full feedback linearized model is obtained

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Dynamic Feedback Linearization

- Considering the very simple stiffness variation model

$$k_i = \tau_{k_i}$$

the dynamics of the system becomes:

$$\begin{bmatrix} \ddot{q} \\ k \end{bmatrix} = \begin{bmatrix} -M^{-1}N \\ 0_{n \times n} \end{bmatrix} + \begin{bmatrix} 0_{n \times n} & -M^{-1}\Phi \\ 0_{n \times n} & I_{n \times n} \end{bmatrix} \begin{bmatrix} \tau \\ \tau_k \end{bmatrix}$$

Problem

The decoupling matrix of the system is structurally singular

Solution

Dynamic extension on the input τ_k is needed

$$\ddot{\tau}_k = u_k$$

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Feedback Linearized Model

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- By defining the control law:

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Control Strategy

- A static state feedback in the state space of the feedback linearized system is used:

$$v_c = \begin{bmatrix} v_q \\ v_k \end{bmatrix}, \quad v_f = \begin{bmatrix} q_d^{[4]} \\ \dot{k}_d \end{bmatrix}$$

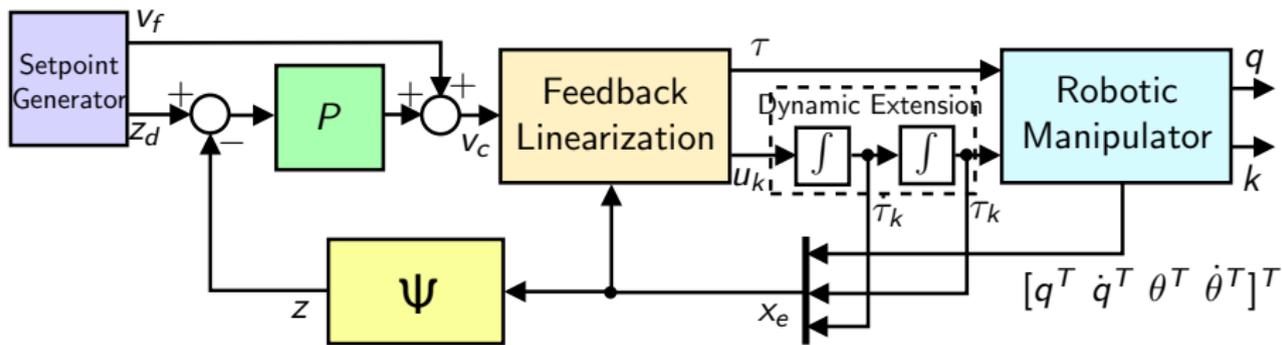
$$z_d = \begin{bmatrix} q_d^T & \dot{q}_d^T & \ddot{q}_d^T & q_d^{[3]T} & k_d^T & \dot{k}_d^T \end{bmatrix}^T$$

- The state vector z of the feedback linearized system and a suitable nonlinear coordinate transformation are defined:

$$z = \begin{bmatrix} q^T & \dot{q}^T & \ddot{q}^T & q^{[3]T} & k^T & \dot{k}^T \end{bmatrix}^T = \Psi(x_e) = \begin{bmatrix} q \\ \dot{q} \\ -M^{-1} [N + \Phi k] \\ -M^{-1} \left[-\dot{M} M^{-1} [N + \Phi k] + \dot{N} + \Phi \dot{k} + \dot{\Phi} k \right] \\ k \\ \dot{k} \end{bmatrix}$$



Control System Architecture



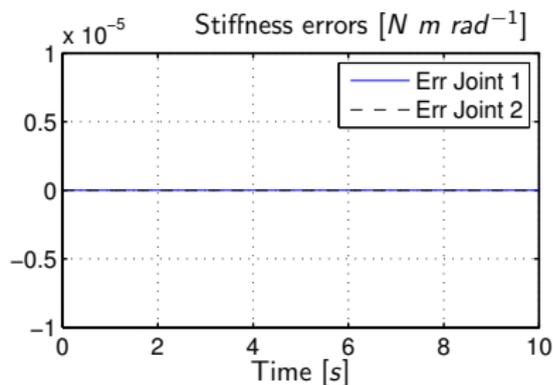
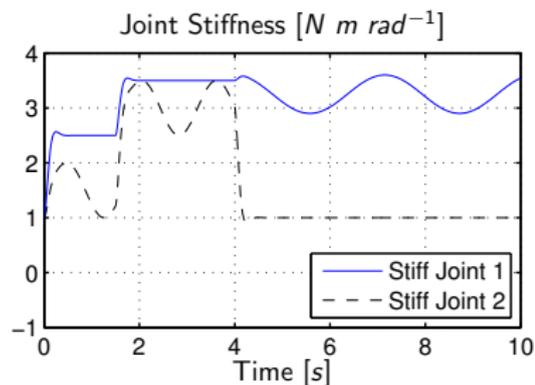
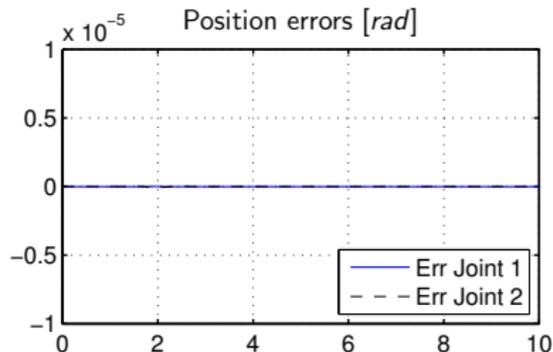
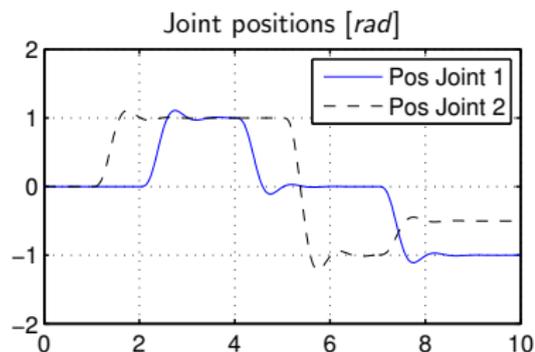
- The controller can be then rewritten as:

$$v_c = v_f + P[z_d - z] = v_f + P[z_d - \Psi(x_e)]$$

where

$$P = \begin{bmatrix} P_{q_0} & P_{q_1} & P_{q_2} & P_{q_3} & 0_{n \times n} & 0_{n \times n} \\ 0_{n \times n} & 0_{n \times n} & 0_{n \times n} & 0_{n \times n} & P_{k_0} & P_{k_1} \end{bmatrix}$$

Simulation of a two-link Planar Manipulator



Application to Antagonistic Variable Stiffness Devices

- Dynamic model of an antagonistic variable stiffness robot

$$M(q) \ddot{q} + N(q, \dot{q}) + \eta_\alpha - \eta_\beta = 0$$

$$B \ddot{\theta}_\alpha + \eta_\alpha = \tau_\alpha$$

$$B \ddot{\theta}_\beta + \eta_\beta = \tau_\beta$$

- By introducing the auxiliary variables

$$p = \frac{\theta_\alpha - \theta_\beta}{2} \quad \text{positions of the generalized joint actuators}$$

$$s = \theta_\alpha + \theta_\beta \quad \text{state of the virtual stiffness actuators}$$

$$F(s) \quad \text{generalized joint stiffness matrix (diagonal)}$$

$$g(q - p) \quad \text{strictly monotonically increasing functions}$$

(generalized joint displacements)

$$h(q - p, s) \quad \text{such that } h_i(0, 0) = 0$$

$$\tau = \tau_\alpha - \tau_\beta, \tau_k = \tau_\alpha + \tau_\beta$$

it is possible to write

$$M(q) \ddot{q} + N(q, \dot{q}) + F(s)g(q - p) = 0$$

$$2B\ddot{p} + F(s)g(p - q) = \tau$$

$$B\ddot{s} + h(q - p, s) = \tau_k$$



Application to Antagonistic Variable Stiffness Devices

- Dynamic model of an antagonistic variable stiffness robot

$$\begin{aligned}M(q) \ddot{q} + N(q, \dot{q}) + \eta_\alpha - \eta_\beta &= 0 \\ B \ddot{\theta}_\alpha + \eta_\alpha &= \tau_\alpha \\ B \ddot{\theta}_\beta + \eta_\beta &= \tau_\beta\end{aligned}$$

- By introducing the auxiliary variables

$$\begin{aligned}p &= \frac{\theta_\alpha - \theta_\beta}{2} && \text{positions of the generalized joint actuators} \\ s &= \theta_\alpha + \theta_\beta && \text{state of the virtual stiffness actuators} \\ F(s) &&& \text{generalized joint stiffness matrix (diagonal)} \\ &&& \text{strictly monotonically increasing functions} \\ g(q - p) &&& \text{(generalized joint displacements)} \\ h(q - p, s) &&& \text{such that } h_i(0, 0) = 0 \\ \tau &= \tau_\alpha - \tau_\beta, \tau_k = \tau_\alpha + \tau_\beta\end{aligned}$$

it is possible to write

$$\begin{aligned}M(q) \ddot{q} + N(q, \dot{q}) + F(s)g(q - p) &= 0 \\ 2B\ddot{p} + F(s)g(p - q) &= \tau \\ B\ddot{s} + h(q - p, s) &= \tau_k\end{aligned}$$



Application to Antagonistic Variable Stiffness Devices

- Dynamic model of an antagonistic variable stiffness robot

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Some Considerations on the Antagonistic Model

- The system is composed by $3N$ rigid bodies (N links and $2N$ actuators)
- The state space dimension is $6N$ (position and velocity of each rigid body)
- The input dimension is $2N$ (actuator torques)
- The output dimension is $3N$ (joint and actuator positions)
- y has dimension $2N$ (position and stiffness of each joint)
- The system has $2N$ DOF (N positioning DOF and N joint stiffnesses DOF)



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Assumptions

- The actuators have uniform mass distribution and center of mass on the rotation axis
- The rotor kinetic energy is due only to their spinning angular velocity
- Each joint is independently actuated by 2 motors in an antagonistic configuration (fully antagonistic kinematic chain)
- Transmission elements with static force-compression characteristic are considered
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Actual Variable Stiffness Joint Implementations

- For antagonistic actuated robot with exponential force/compression characteristic (Palli et al. 2007)



$$\begin{aligned}f_i(s_i) &= e^{a s_i} \\g_i(q_i - p_i) &= b \sinh(c (q_i - p_i)) \\h_i(q_i - p_i, s_i) &= d [\cosh(c (q_i - p_i)) e^{a s_i} - 1]\end{aligned}$$

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- For the variable stiffness actuation joint (VSA), using a third-order polynomial approximation of the transmission model (Boccardo, Bicchi et al. 2006)



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State reconstruction

- All the state variables are known (full state feedback)



State Reconstruction

The whole state of the system can be reconstructed by means of:

- State Observers
 - ▶ Increase the complexity of the system
 - ▶ Parameters adaptation is needed
 - ▶ Require a measure (or a estimation) of the external forces
- Filtering of position information
 - ▶ Generates noisy velocity signals
 - ▶ High-speed acquisition and computation system
- Tachometers
 - ▶ Increase costs
 - ▶ Difficulties due to the integration into the system



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Disturbance decoupling problem

- The vector relative degrees of the outputs with respect to the input w is:

$$\begin{aligned} L_d q &= 0_{N \times N} \quad , \quad L_d F(s) = 0_{N \times N} \\ L_d L_f q &= M(q)^{-1} \quad , \quad L_d L_f F(s) = M(q)^{-1} \frac{\partial g(q-p)}{\partial q} \end{aligned}$$

- The disturbance decoupling problem can't be solved
- The joint positions can't be decoupled from the disturbance
 - ▶ The external load can be compensated only in steady state conditions
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External load estimation

- The dynamic equation of the robot manipulator can be rewritten to take into account for external load

$$M(q)\ddot{q} + N(q, \dot{q}) + \eta_\alpha - \eta_\beta = \tau_{\text{ext}}$$

- The generalized momenta of the robotic arm is:

$$p = M(q)\dot{q}$$

$$\dot{p} = \dot{M}(q)\dot{q} + M(q)\ddot{q} = \dot{M}(q)\dot{q} - N(q, \dot{q}) - \eta_\alpha + \eta_\beta + \tau_{\text{ext}}$$

- Recalling the general property

$$q^T [\dot{M}(q) - 2C(q, \dot{q})]q = 0 \Rightarrow \dot{M}(q) = C(q, \dot{q}) + C^T(q, \dot{q})$$

we obtain

$$\dot{p} = -C^T(q, \dot{q})\dot{q} - D\dot{q} - g(q) - \eta_\alpha + \eta_\beta + \tau_{\text{ext}} = \bar{N}(q, \dot{q}) - \tau_q + \tau_{\text{ext}}$$

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External load estimation

- Defining the external load estimation as:

$$\hat{\tau}_{ext} = L \left[\int (\tau_q - \bar{N}(q, \dot{q}) - \hat{\tau}_{ext}) dt + p \right]$$

whit positive defined (diagonal) L , the torque extimation dynamic is:

$$\dot{\hat{\tau}}_{ext} = -L\hat{\tau}_{ext} + L\tau_{ext}$$

- It is possible to define the transfer function between the real and the observed external torques:

$$\hat{\tau}_{ext_i}(s) = \frac{L_i}{s + L_i} \tau_{ext_i}(s)$$

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- The actuators have uniform mass distribution and center of mass on the rotation axis
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Feedback linearization problem

The sum of the vector relative degrees is not equal to the state dimension

The full state linearization problem can't be solved

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Feedback linearization of the complete model

- Complete dynamic model of the antagonistic actuated arm:

$$\begin{aligned}M(q)\ddot{q} + H_\alpha \ddot{\theta}_\alpha + H_\beta \ddot{\theta}_\beta + N(q, \dot{q}) + \eta_\alpha - \eta_\beta &= \tau_{ext} \\ B\ddot{\theta}_\alpha + H_\alpha^T \ddot{q} + \eta_\alpha &= \tau_{\theta_\alpha} \\ B\ddot{\theta}_\beta + H_\beta^T \ddot{q} + \eta_\beta &= \tau_{\theta_\beta}\end{aligned}$$

with upper-triangular matrices H_α and H_β

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Analysis of different configurations

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Conclusions

- The feedforward control action needed to perform a desired motion profile has been computed
- The feedback linearization problem with decoupled control has been solved taking into account different stiffness variation models
- The simultaneous asymptotic trajectory tracking of both the position and the stiffness has been achieved by means of an outer linear control loop
- These results can be easily extended to the mixed rigid/elastic case
- The proposed approach has been used to model several actual implementation of variable stiffness devices

Questions?

Thank you for your attention...



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