Input-Output Finite-Time Stabilization with Constrained Control Inputs

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Motivations

Input-output finite-time stability vs classic IO stability

IO stability

A system is said to be IO \mathcal{L}_p -stable if for any input of class \mathcal{L}_p , the system exhibits a corresponding output which belongs to the same class

IO-FTS

A system is defined to be IO-FTS if, given a class of norm bounded input signals over a specified time interval T, the outputs of the system do not exceed an assigned threshold during T

- Motivations

Main features of IO-FTS

IO-FTS:

- involves signals defined over a finite time interval
- does not necessarily require the inputs and outputs to belong to the same class
- specifies a *quantitative* bounds on both inputs and outputs
- IO stability and IO-FTS are independent concepts

- Motivations

Contribution of the paper

- In this paper we provide extend the *classical* definition of IO-FTS to the one of structured IO-FTS
- Structured IO-FTS permits to incorporate amplitude constraints on the control input variables in the definition of the stabilization problem
- A necessary and sufficient condition is given for the solution of the IO finite-stabilization problem, when the input signals belong to L₂
- A sufficient condition is given for the solution of the IO finite-stabilization problem, when the inputs belong to L_∞

- Notation

Notation

- \mathcal{L}_p denotes the space of vector-valued signals whose *p*-th power is absolutely integrable over $[0, +\infty)$
- The restriction of \mathcal{L}_p to $\Omega := [t_0, t_0 + T]$ is denoted by $\mathcal{L}_p(\Omega)$
- Given the time interval Ω , a symmetric positive definite matrix-valued function $R(\cdot)$, bounded on Ω , and a vector-valued signal $s(\cdot) \in \mathcal{L}_p(\Omega)$, the weighted signal norm

$$\left(\int_{\Omega}\left[s^{\mathsf{T}}(\tau)R(\tau)s(\tau)\right]^{\frac{p}{2}}d\tau\right)^{\frac{1}{p}}$$

will be denoted by $||s(\cdot)||_{p,R}$. If $p = \infty$

$$\|s(\cdot)\|_{\infty,R} = \operatorname{ess\,sup}_{t\in\Omega} [s^{T}(t)R(t)s(t)]^{\frac{1}{2}}$$

Problem Statement

Structured IO-FTS of LTV systems

• Let
•
$$\mathcal{W}$$
 be a class of input signals defined over $\Omega = [t_0, t_0 + T]$
• $Q(t) := \operatorname{diag}(Q_1(t), \dots, Q_{\alpha}(t))$, with $Q_i(t) \in \mathbb{R}^{m_i \times m_i}$,
 $i = 1, \dots, \alpha$, a positive definite matrix-valued function
• The system
 $\dot{x}(t) = A(t)x(t) + G(t)w(t), \quad x(t_0) = 0$ (1a)
 $y(t) = C(t)x(t) + F(t)w(t)$ (1b)
is said to be structured IO-FTS with respect to $(\mathcal{W}, Q(\cdot), \Omega)$ if
 $w(\cdot) \in \mathcal{W} \Rightarrow y_i^T(t)Q_i(t)y_i(t) < 1, \quad t \in \Omega,$
 $i = 1, \dots, \alpha,$
where the output vector $y(t)$ is partitioned as follows
 $y(t) = (y_1^T(t) \cdots y_{\alpha}^T(t))^T, \quad t \in \Omega.$

Problem Statement

The finite-time stabilization problem

In the finite-time stabilization problem we consider the LTV system

$$\dot{x}(t) = A(t)x(t) + B(t)u(t) + G(t)w(t), \quad x(t_0) = 0$$
(2a)
$$y(t) = C(t)x(t) + F(t)w(t)$$
(2b)

where $u(\cdot): \Omega \mapsto \mathbb{R}^q$ is the control input and $w(\cdot)$ is the disturbance (exogenous) input

Similarly to what has been done for the output, the control input vector u(t) is partitioned as

$$u(t) = \left(u_1^T(t)\cdots u_{\beta}^T(t)\right)^T$$

Problem Statement

The IO finite-time stabilization problem via state feedback - 1

• Consider β positive definite weighting matrix-valued functions $T_i(t) \in \mathbb{R}^{q_i \times q_i}, i = 1, \dots, \beta$, and define $T(t) := \operatorname{diag}(T_1(t), \ldots, T_{\beta}(t))$

Problem Statement

The IO finite-time stabilization problem via state feedback - 2

Given a positive scalar T, the class of signals \mathcal{W} , and the weighting matrices $Q(\cdot)$, $T(\cdot)$, find a state feedback control law u(t) = K(t)x(t).where $K(\cdot)$: $\Omega \mapsto \mathbb{R}^{q \times n}$, such that the system $\dot{x}(t) = (A(t) + B(t)K(t))x(t) + G(t)w(t)$ (3a) $\begin{pmatrix} y(t) \\ u(t) \end{pmatrix} = \begin{pmatrix} C_{1}(t) \\ \vdots \\ C_{\alpha}(t) \\ K_{1}(t) \\ \vdots \\ K_{\beta}(t) \end{pmatrix} \times (t) + \begin{pmatrix} F_{1}(t) \\ \vdots \\ F_{\alpha}(t) \\ 0 \end{pmatrix} w(t)$ (3b)

is structured IO-FTS with respect to $(\mathcal{W}, \text{diag}(Q(\cdot), T(\cdot)), \Omega)$.

Problem Statement

The IO finite-time stabilization problem via state feedback - 3

Note that the partition

$$T(t) := \mathsf{diag}(T_1(t), \dots, T_eta(t))$$

induces the following structure for the controller gain

$$\mathcal{K}(t) = \left(\mathcal{K}_{1}^{T}(t) \cdots \mathcal{K}_{\beta}^{T}(t)\right)^{T}, \quad t \in \Omega.$$
(4)

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Structured IO-FTS

Problem Statement

Considered class of input signals

\mathcal{W}_2 signals

Norm bounded square integrable signals over Ω , defined as follows

$$\mathcal{W}_2\left(\Omega\,,R(\cdot)
ight):=\{w(\cdot)\in\mathcal{L}_2(\Omega)\,:\,\|w\|_{2,R}\leq 1\}\;.$$

\mathcal{W}_∞ signals

Uniformly bounded signals over Ω , defined as follows

 $\mathcal{W}_{\infty}\left(\Omega\,, R(\cdot)
ight):=\left\{w(\cdot)\in\mathcal{L}_{\infty}(\Omega)\,:\, \|w\|_{\infty,R}\leq 1
ight\}.$

Previous results

The analysis results presented in **Amato et al., Automatica 2010** and **Amato et al., TAC 2012** have been extended to the case of structured IO-FTS



F. Amato et al.

Input-output Finite-Time Stabilization of Linear Systems *Automatica*, 2010

F. Amato et al.

Input-Output Finite-Time Stability of Linear Systems: Necessary and Sufficient Conditions

IEEE Transactions on Automatic Control, 2012

Proper and strictly-proper linear systems

For the class of W_2 signals we consider a strictly proper system, i.e. $F(\cdot) = 0$, otherwise the concept of structured IO-FTS would be **ill-posed**.

 W_2 includes signals that are unbounded on a zero measure interval included in Ω . For those signals, if $F(\cdot) \neq 0$ then there exists at least one time instant where the output would be unbounded

For the class of \mathcal{W}_{∞} signals we consider proper system, i.e. $F(\cdot) \neq 0$

Structured IO-FTS for W_2 signals

Given system (1) with $F(\cdot) = 0$, the class of inputs W_2 , a continuous positive definite matrix-valued function $Q(\cdot)$, and the time interval Ω , the following statements are equivalent:

i) System (1) is structured IO-FTS with respect to $(\mathcal{W}_2, Q(\cdot), \Omega)$.

ii) The inequality

$$\lambda_{\max} \left(Q_i^{\frac{1}{2}}(t) C_i(t) W(t, t_0) C_i^{T}(t) Q_i^{\frac{1}{2}}(t) \right) < 1$$
(5)

holds for all $t \in \Omega$ and $i = 1, ..., \alpha$, where $W(\cdot, \cdot)$ is the positive semidefinite solution of the DLE

$$\dot{W}(t, t_0) = A(t)W(t, t_0) + W(t, t_0)A^T(t) + G(t)R(t)^{-1}G^T(t)$$
(6a)

$$W(t_0, t_0) = 0$$
(6b)

iii) The coupled DLMI/LMI

$$\begin{pmatrix} \dot{P}(t) + A^{T}(t)P(t) + P(t)A(t) & P(t)G(t) \\ G^{T}(t)P(t) & -R(t) \end{pmatrix} < 0$$
 (7a)

$$P(t) \ge C_i^T(t)Q_i(t)C_i(t), \quad i = 1, \dots, \alpha,$$
(7b)

admits a positive definite solution $P(\cdot)$ over Ω .

Structured IO-FTS for \mathcal{W}_{∞} signals

Let $\widetilde{Q}_i(t) = (t - t_0)Q_i(t)$; if there exist a positive definite and continuously differentiable matrix-valued function $P(\cdot)$ and α scalar functions $\theta_1(\cdot), \ldots, \theta_{\alpha}(t) > 1$ such that the coupled DLMI/LMI

$$\begin{pmatrix} \dot{P}(t) + A^{T}(t)P(t) + P(t)A(t) & P(t)G(t) \\ G^{T}(t)P(t) & -R(t) \end{pmatrix} < 0,$$

$$(8a)$$

$$\dot{P}_{i}(t)R(t) - R(t) \ge 2\theta_{i}(t)F_{i}^{T}(t)Q_{i}(t)F_{i}(t),$$

$$i = 1, \dots, \alpha, \qquad (8b)$$

$$P(t) \ge 2 \theta_i(t) C_i(t)^T \widetilde{Q}_i(t) C_i(t), \quad i = 1, \dots, \alpha,$$
(8c)

are fulfilled over Ω , then system (1) is IO-FTS with respect to $(\mathcal{W}_{\infty}, Q(\cdot), \Omega)$.

Synthesis Results

Theorem 1

IO finite-time stabilization for \mathcal{W}_2 signals

Given the class of disturbances W_2 and $F(\cdot) = 0$, the IO finite-time stabilization problem via state feedback is solvable if and only if there exist a positive definite and continuously differentiable matrix-valued function $\Pi(\cdot)$, and β continuously differentiable matrix-valued functions $L_1(\cdot), \ldots, L_{\beta}(\cdot)$ such that,

$$\begin{pmatrix} \Theta(t) & G(t) \\ G^{T}(t) & -R(t) \end{pmatrix} < 0,$$

$$\begin{pmatrix} \Pi(t) \\ C_{i}(t)\Pi(t) \\ \Xi_{i}(t) \end{pmatrix} \geq 0, \quad i = 1, \dots, \alpha$$

$$(9b)$$

$$\begin{pmatrix} \Pi(t) & L_j^{\mathsf{T}}(t) \\ L_j(t) & \Upsilon_j(t) \end{pmatrix} \ge 0, \qquad j = 1, \dots, \beta$$
(9c)

for all $t \in \Omega$, with

$$\Theta(t) := -\dot{\Pi}(t) + \Pi(t)A^T(t) + A(t)\Pi(t) + B(t)\left(L_1^T(t)\cdots L_\beta^T(t)\right)^T + \left(L_1^T(t)\cdots L_\beta^T(t)\right)B^T(t),$$

 $\Xi_i(t) := Q_i^{-1}(t)$, and $\Upsilon_j(t) := T_i^{-1}(t)$.

The controller gain which solves the IO finite-time stabilization problem via state feedback is given by (4) with $K_i(t) = L_i(t)\Pi^{-1}(t), j = 1, \dots, \beta$.

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Synthesis Results

L Theorem 1

Sketch of proof - 1

Conditions (7) for the augmented output closed-loop system (3) read

$$\begin{pmatrix} \dot{P}(t) + A_{cl}^{\mathsf{T}}(t)P(t) + P(t)A_{cl}(t) & P(t)G(t) \\ G^{\mathsf{T}}(t)P(t) & -R(t) \end{pmatrix} < 0,$$
(10a)

$$P(t) \ge C_i^{T}(t)Q_i(t)C_i(t), \quad i = 1, \dots, \alpha$$

$$P(t) \ge K_j^{T}(t)T_j(t)K_j(t), \quad j = 1, \dots, \beta,$$
(10b)
(10c)

where

$$A_{cl}(\cdot) = A(\cdot(+B(\cdot)K(\cdot)$$

Synthesis Results

L Theorem 1

Sketch of proof - 2

Let $\Pi(t) = P^{-1}(t)$. By pre- and post-multiplying (10a) by $\begin{pmatrix} \Pi(t) & 0 \\ 0 & I \end{pmatrix} > 0$, and by pre- and post-multiplying (10b) and (10c) by $\Pi(t)$, we have $\begin{pmatrix} -\Pi(t) + \Pi(t)A_{cl}^{T}(t) + A_{cl}(t)\Pi(t) & G(t) \\ G^{T}(t) & -R(t) \end{pmatrix} < 0,$ (11a) $\left(egin{array}{cc} \Pi(t) & \Pi(t)C_i^{ op}(t) \ C_i(t)\Pi(t) & \Xi_i(t) \end{array}
ight)\geq 0\,,\quad i=1\,,\dots\,,lpha$ (11b) $\begin{pmatrix} \Pi(t) & \Pi(t) \mathcal{K}_j^{\mathsf{T}}(t) \\ \mathcal{K}_i(t) \Pi(t) & \Upsilon_i(t) \end{pmatrix} \ge 0, \quad j = 1, \dots, \beta$ (11c)

where (11b) and (11c) are obtained by applying the Schur complements. The proof of the theorem then readily follows by letting $L_j(t) = K_j(t)\Pi(t)$ for $j = 1, ..., \beta$. Synthesis Results

Theorem 2

IO finite-time stabilization for \mathcal{W}_{∞} signals

Given the class of disturbances W_{∞} , the IO finite-time stabilization problem via state feedback is solvable if there exist a positive definite and continuously differentiable matrix-valued function $\Pi(\cdot)$, β continuously differentiable matrix-valued functions $L_1(\cdot), \ldots, L_{\beta}(\cdot)$, and α strictly positive functions $\lambda_1(\cdot), \ldots, \lambda_{\alpha}(\cdot) < 1$ such that (9a) and

$$R(t) - \lambda_i(t)R(t) \ge 2F_i^T(t)Q_i(t)F_i(t),$$

$$i = 1, \dots, \alpha$$
(12a)

$$\begin{pmatrix} \Pi(t) & \Pi(t)C_i^T(t) \\ C_i(t)\Pi(t) & \frac{\lambda_i(t)}{2} \widetilde{\Xi}_i(t) \end{pmatrix} \ge 0, \quad i = 1, \dots, \alpha$$
(12b)
$$\begin{pmatrix} \Pi(t) & L_i^T(t) \\ 0 & 0 & 0 \end{pmatrix} \ge 0, \quad i = 1, \dots, \alpha$$
(12c)

$$\begin{pmatrix} \Pi(t) & L_j(t) \\ L_j(t) & \widetilde{\Upsilon}_j(t) \end{pmatrix} \ge 0, \quad j = 1, \dots, \beta$$
(12c)

hold, when $t \in \Omega$, with $\widetilde{\Xi}_i(t) := ((t - t_0)Q_i(t))^{-1}$, and $\widetilde{\Upsilon}_j(t) := ((t - t_0)T_j(t))^{-1}$. The controller gain which solves the IO finite-time stabilization problem via state feedback is given by (4) with $K_j(t) = L_j(t)\Pi^{-1}(t), j = 1, \dots, \beta$.

Quarter car suspension model

- *M_s* sprung mass
- *M_u* unsprung mass
- B_s suspension damping coefficient
- K_s suspension spring elastic coefficient
- K_u elastic coefficient that models tire deflection
- *u_f* active force generated by the hydraulic actuator S



Figure: Schematic representation of the active suspension system.

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Model

Letting

- x_s and x_u the vertical displacement of the sprung and unsprung masses, respectively
- x_o the vertical ground displacement caused by road unevenness

and choosing as state variables

- the suspension stroke $x_s x_u$
- the tire deflection $x_u x_o$

and their derivatives The resulting open-loop dynamical model reads

$$\dot{x}(t) = \begin{pmatrix} 0 & 1 & 0 & -1 \\ -\frac{K_s}{M_s} & -\frac{B_s}{M_s} & 0 & \frac{B_s}{M_s} \\ 0 & 0 & 0 & 1 \\ \frac{K_s}{M_u} & \frac{B_s}{M_u} & -\frac{K_u}{M_u} & -\frac{B_s}{M_u} \end{pmatrix} x(t) + \begin{pmatrix} 0 \\ \frac{u_{\max}}{M_s} \\ 0 \\ -\frac{u_{\max}}{M_u} \end{pmatrix} u(t) + \begin{pmatrix} 0 \\ 0 \\ -1 \\ 0 \end{pmatrix} w(t), \quad (13)$$

where the normalized active force $u(\cdot) = u_f(\cdot)/u_{\text{max}}$ is the control input and the exogenous input $w(\cdot) = \dot{x}_o(\cdot)$ represents the disturbance caused by the road roughness.

Design constraints

When designing a controller a number of constraints should be considered.

• To ensure a firm uninterrupted contact of wheels to road, the dynamic tire load should not exceed the static one

$$k_u |x_3(t)| < (m_s + m_u) g \quad \forall t \ge 0.$$
 (14)

The suspension stroke should fulfill the following constraint

$$|x_1(t)| \leq SS, \quad \forall \ t \geq 0.$$
 (15)

In order to cast the control design problem in the IO-FTS framework, we consider the following system outputs

$$\begin{pmatrix} y_1(t) \\ y_2(t) \\ y_3(t) \end{pmatrix} = \begin{pmatrix} \dot{x}_2(t) \\ \frac{x_1(t)}{5S} \\ \frac{k_u x_3(t)}{g(m_s + m_u)} \end{pmatrix} = \begin{pmatrix} C_1 \\ C_2 \\ C_3 \end{pmatrix} x(t) + \begin{pmatrix} D_1 \\ D_2 \\ D_3 \end{pmatrix} u,$$
(16)

where

$$C_{1} = \left(-\frac{k_{s}}{m_{s}} - \frac{c_{s}}{m_{s}} \ 0 \ \frac{c_{s}}{m_{s}}\right), \qquad D_{1} = \frac{u_{\max}}{m_{s}},$$

$$C_{2} = (1 \ 0 \ 0 \ 0), \qquad D_{2} = 0,$$

$$C_{3} = (0 \ 0 \ 1 \ 0), \qquad D_{3} = 0.$$

Model parameters

The following values for the model parameters have been taken from Chen and Guo, TCST, 2005

$$M_{s} = 320 \ kg , \qquad K_{s} = 18 \ \frac{kN}{m} , \qquad K_{s} = 18 \ \frac{kN}{m} , \qquad K_{s} = 18 \ \frac{kN}{m} , \qquad K_{u} = 200 \ \frac{kN}{m} , \qquad M_{u} = 40 \ kg , \qquad u_{max} = 1.5 \ kN , \qquad SS = 0.08 \ m , \qquad K_{u} = 200 \ \frac{kN}{m} , \qquad K_{u} = 1.5 \ kN , \qquad K_{u} = 1.5 \ k$$

1 ...

Actuator saturation

Due to actuator saturation, the active force is bounded by u_{max} , i.e. the normalized force has to satisfy

$$|u(t)| \leq 1, \quad \forall t \geq 0.$$
(17)

In order to frame the problem of designing the active suspension control system in the context of structured IO finite-time stabilization let us rewrite the output equation as

$$\begin{pmatrix} y_{1}(t) \\ y_{2}(t) \\ y_{3}(t) \\ u(t) \end{pmatrix} = \begin{pmatrix} \dot{x}_{2}(t) \\ \frac{x_{1}(t)}{SS} \\ \frac{k_{u}x_{3}(t)}{\overline{g}(m_{s}+m_{u})} \\ \mathcal{K}(t)x(t) \end{pmatrix} = \begin{pmatrix} C_{1} + D_{1}\mathcal{K}(t) \\ C_{2} + D_{2}\mathcal{K}(t) \\ C_{3} + D_{3}\mathcal{K}(t) \\ \mathcal{K}(t) \end{pmatrix} x(t) .$$
(18)

Reference disturbance

We will design the time-varying state feedback K(t) that optimize the response to an isolated bump modeled as the W_2 disturbance

$$w(t) = \begin{cases} \frac{M}{2} \left(1 - \cos\left(\frac{2\pi V}{L}t\right) \right), & 0 \le t \le \frac{L}{V} \\ 0, & t > \frac{L}{V} \end{cases}$$
(19)

where $M = 0.1 \, m$, $L = 5 \, m$ are the bump height and width, respectively, while $V = 45 \, km/h$ is the vehicle forward velocity.

In particular, given the bump (19) we want to minimize the body acceleration $y_1(t) = \dot{x}_2(t)$ fulfilling the constraints (14)–(17).

- Example

IO-FTS parameters

We consider the following IO-FTS parameters

$$T=2 \ s$$
, $R=8$.

Furthermore, given the selected outputs (18), the two outputs weighting matrices

$$Q_2=Q_3=1$$
 ,

allows to take into account the constraints (14) and (15), while the input weighting matrix is

$$T_1 = 0.15$$

which allows to exploit the full scale of the control input when (19) is considered.

In order to minimize the body acceleration it is possible to exploit Theorem 1 and solve the following optimization problem

 $\begin{array}{l} \text{minimize } \Xi_1 \\ \text{subject to (9)} \end{array}$ (20)

where $\Xi_1 = Q_1^{-1}$.

- Example

Solving the problem

- Assuming the two matrix-valued functions Π(·) and L(·) to be piecewise linear, it is possible to recast problem (20) in the LMIs framework.
- By solving (20), we get Ξ_{1_{min}} = 7.22 and the two feasible matrix-valued functions Π(·) and L(·); the time-varying controller K(t) is then given by K(t) = L(t)Π(t)⁻¹.

Results - 1



Figure: Bump response: IO-FTS time-varying controller (–), constrained \mathcal{H}_{∞} controller (– –, Chen and Guo, TCST, 2005).

Results - 2



Figure: Bump response: time behavior of the weighted outputs $y_2(t)^T Q_2 y_2(t)$ and $y_3(t)^T Q_3 y_3(t)$ when the IO-FTS time-varying controller is considered.

Conclusions

- The notion of structured IO-FTS has been introduced
- Structured IO-FTS allows to take into account design constraint on the control input
- Conditions for IO finite-time stabilization (in the *structured context*) of LTV systems via state feedback have been given
- The effectiveness of the approach has been illustrated by means of an engineering case-study

Thank you!